PEAPOD

<u>P</u>neumatically <u>E</u>nergized <u>A</u>uto-throttled <u>P</u>ump for a <u>D</u>evelopmental Upperstage

Manufacturing Status Review



Customer: Special Aerospace Services Chris Webber and Tim Bulk







- Project Overview
- Pump Manufacturing
- Pump Control
 - Valving
 - Electronics
 - Software
- Budget
- Conclusion







- Design and manufacture a pneumatically powered pump system for use on an upper stage rocket engine or lander.
 - Proof of concept pump system for hypergolic propellants
 - 10%-100% throttleability
 - Pneumatically powered







Level	Functional Success	Performance Success	Functional Requirement
1	 Pneumatic power Digital control Meets safety requirements 	 750 ± 15 psi outlet pressure Structural FOS 2.5 120 seconds of operation 75% efficiency of pump at full throttle 	 FR1 – System is pneumatically driven FR7 - FOS of 2.5 FR8 – 75% efficiency at full throttle FR3 – Pump outlet is at 750 ± 15 PSI
2	 Propellant stream throttling All level 1 requirements 	 10-100% throttleability 0-100% throttle in 2 seconds All level 1 requirements 	 FR2 – Pump system is throttleable FR4 – Pump system can run through throttle profile FR5 – Pump is restartable
3	 Hypergolic compatible All level 1 and 2 requirements 	0-100% throttle in 1 secondAll level 1 and 2 requirements	• FR6 – System is built to hypergolic standards
Overvi	ew Pump Manufacturing	Pump Control Testi Sched	ng/ lule Budget



Functional Block Diagram



<u>Subsystems</u> 1. Pump 2. Drive System 3. Control 4. Test













Current Schedule





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Current Schedule









- What is included:
 - Housing
 - Gears
 - Driveshaft
 - Bearings
 - Seals
 - Assembly
 - Helium Regulator Holster







- Gear pump is made up of:
 - Gears To be Manufactured
 - Housing To be Manufactured
 - Close-out panel To be Manufactured
 - Driveshaft To be finalized, will be keyed and altered
 - Alignment Shaft To be finalized, will be keyed and altered
 - Seals Purchased; Geometry is being triple checked
 - Main bearings Purchased; Thrust and roller bearings to be purchased
 - Coupler To be Purchased
 - Collar To be Purchased







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- Wire EDM profile with ground faces
- Finalizing stock requirements (17-4 PH Stainless)
- CFD results may dictate minor changes
- No post machining with decided manufacturer
 - Delivery 4 weeks ARO











- CNC here at CU, use a rough cut first to minimize time on CNC
- Benefit working from our model

Pump

Manufacturing

- Based off of continued CFD results, a second housing with looser tolerances may be made – perhaps here or at SAS
 - This gives us a back up
 - Must continue to move forward and complete CFD concurrently

Pump

Control

- <u>Time frame: 4 weeks</u>
- Concerns: Depth of extruded cut, maintaining the profile, materials, special tools to manufacture

Budget

Testing/

Schedule













- CNC here at CU
- Based off of continued CFD results, a second housing with looser tolerances may be made – perhaps here or at SAS
 - This gives us a back up, singular plate may work for both housings TBD
- Time frame: 4 weeks lead time
- Drawing to be finalized: still awaiting relief grooves, seal groove geometry may change for optimal performance
- Concerns: material, special tools to manufacture, rotary shaft seal and geometry

















- Seals
 - 1 large O-ring for faceplate and housing
 - 1 O-ring shaft seal
 - Groove geometry is being triple checked; May utilize a double seal
 - Ready for pick-up in Denver
- Bearings
 - Bearing manufacturer that commonly supplies bearings for gear pumps is sending 'samples' for free (shipped, awaiting arrival)
 - This is putting constraints on our shafts metric options may solve this
 - Thrust bearings are outside of working fluid and work with a collar to retain the drive shaft
 - Roller bearings are included to avoid wear on the inside of the housing and necessitate post-machining of the shafts





- Driveshaft and placement
 - Post-machine here
 - We can key in the shop here, allows for proper contact and sizing to maintain integrity of housing
 - Metric options are being explored to obtain proper undersizing for the performancing sleeve bearings
 - Timeframe: 20 hours





- Seal groove geometry will be finalized by Tuesday COB
- Meeting with Matt Rhode Tuesday to nail down logistics proper material and special tools
- Order and get materials and tools by Feb. 13th
- Start machining ASAP Feb 13th or 14th





Pump Assembly







Pump Assembly







Pump Assembly







Timeframe: 1 week for assembly







Shaft



Timeframe: 1 week for assembly







- Constructing an Electronic Helium Regulator
 - To control the drive system
- Manufacture a holster to house the stepper motor
- Timeframe: 80 hours





Electronic He Regulator



- Solenoid regulator was too expensive
- Manufacturing from aluminum
- Houses stepper motor
- Need 8 Nm of torque to turn the pressure regulator at its lowest settings
- Similar setup for back pressure regulator



Pump Control















Signal Conditioning

- Design will be tested in a breadboard
- PCB board will be designed based on the breadboard
- PCB board will be populated
- Testing
 - Component testing will be done on the breadboard
 - Integration will be done with the PCB board







- Electronics
 - Sensors
 - DAQs
 - NI 9211 (temperature readings)
 - NI 9205 (pressure readings, similar to NI 6002 but more powerful)
 - will use NI 6002 for final tests
 - 100 hours left for all testing and integration









Software

- LabVIEW
 - Development order:
 - File system complete
 - Component tests/DAQ system in progress (~10 hours, <1 week)
 - Control systems model, simulate, implement in progress (~45 hours, 4 weeks)
 - Full system tests TBD (~60 hours, 5 weeks)
- DAQ Software Drivers
 - Already have
- Post test analysis MATLAB (file read functions complete)







- Control code overview:
 - Safety implicit to error check
 - Throttle manual or automatic

Overview

• Backpressure faster than power - prevent oscillation





Overview





- Pressure valve control:
 - Some data kept in memory – check valve position

Pump

Manufacturing









• Backpressure control:

Pump

Manufacturing

• Fastest control component

Overview









Software – thermocouple test block diagram









• Software – thermocouple test user interface











- Combined pressure and temperature user interface
- 115 hours remain



Pump




Testing Timeline



Component Testing – QC and Calibration

Common ant	Testing to be Conducted	Man Hauna	Statura	1 iming	
Component	Testing to be Conducted	Man nours	Status	Personnel:	
Pressure Transducer (Air P1, Water P1)	Calibration through ranges of known shop air pressures	6 Hours : 1.5*(2*1 + 2*1 + 2*1)	50%	Operators required	
Air Pressure Regulator (EPR)	Operation through 0-110PSI to calibrate outlet pressure/thread engagement	10.5 Hours : 3*1 + 3*1.5 + 3*1	0%	Hours: Time required to prepare and	
Water Back Pressure Regulator (BPR)	Operation through 0-750PSI to calibrate inlet pressure/thread engagement	10.5 Hours : 3*1 + 3*1.5 + 3*1	0%	assemble test	
Thermocouples (T1, T2)	Calibration at known temperatures (Ice bath, boiling)	4 Hours : 2*0.5 + 2*1 + 2*0.5	100%	required to conduct test	
Manual Ball Valve (Water MBV1)	Verify correct operation (on/off) and no leaking	25 Minutes : $1*0.2 + 1*0.1 + 1*0.2$	0%	Hours: Time	
Solenoid Valve (Water SV1, Air SV2)	Correct opening and closing upon energization and de-energization	1.5 Hours : 2*0.5 + 2*0.25 + 2*0.5	0%	required to disassemble test and save results	
Tachometer	Calibration at known RPMs	6 Hours : 2*1 + 2*1 + 2*1	0%	Multiplier:	
Manual Air Regulator (MR1)	Calibration to ranges 100-125 with known shop air pressures	7 Hours : 2*1.5 + 2*1 + 2*1	0%	Repeat test for identical sensor /component	
Full Air Assembly	Torque Verification @ RPM, conducted with friction test, fan test	13.5 Hours : 3*2 + 3*1.5 + 3*1	0%	, component	

*Subsystem Testing included in backups

Total Man Hours: 113.4









Integration ³⁸











Total	\$ 7,513.74
Budget	8000
margin	6.078%

Progress	73%	(by part #)
	34%	(by \$ Value)





Received

Ordered

Part

part number price



Total

sub total

Quantity

shipping



x	х	Globe Motor	VA10J	\$ 1,238.00	1	\$ 1,238.00	\$ 125.00	\$ 1,363.0	0								3	2	
x	x	Tachometer	A2108	\$ 349.00	1	\$ 349.00	\$-	\$ 349.0	0										
x	х	Reflective Tape	205t	\$ 5.00	1	\$ 5.00	\$-	\$ 5.0	0										
x	x	reflective tab	205ts	\$ 5.00	1	\$ 5.00	\$-	\$ 5.0	0										
	x	Water pipe	t53480	\$ 29.44	1	\$ 29.44	\$ 21.90	\$ 51.3	4										
	x	Hose2Pipe adp	4kg88	\$ 4.81	2	\$ 9.62	\$ 5.00	\$ 14.6	2										
x	х	Air Regulator	21U842	\$ 228.00	1	\$ 228.00	\$ 33.21	\$ 261.2	1										
x	х	Pipe tee	4429K229	\$ 14.27	2	\$ 28.54	\$ 15.30	\$ 43.8	4 Received	Ordered	Part	part numbe	r price	Quantity	sub tota	1 1	shipping	Total	٦
x	х	Air Filter	3248T11	\$ 78.10	1	\$ 78.10	\$-	\$ 78.1	0		Tooling for gea	ars	\$ 500.00) 1	\$ 500	.00	\$ 75.00	\$ 575.00	ว
x	х	lubricator	8520T19	\$ 82.58	1	\$ 82.58	\$-	\$ 82.5	8		Water drum		\$ 500.00) 1	\$ 500	.00	\$ 75.00	\$ 575.00	5
x	х	lube	1298K2	\$ 24.47	1	\$ 24.47	\$-	\$ 24.4	7		Teflon Seal		\$-	(\$	-	0	\$ -	
	х	steel pipe	4457K42	\$ 14.72	1	\$ 14.72	\$ 27.21	\$ 41.9	3		binding report		\$ 100.00) 1	\$ 100	.00	0	\$ 100.00	3
	x	pipe nipple	4830K194	\$ 3.93	1	\$ 3.93	\$-	\$ 3.9	3 x	x	Microsoft Offic	ce	\$ -	1	\$	-	0	\$ -	
	х	Brass Valve	4629K44	\$ 18.53	1	\$ 18.53	\$-	\$ 18.5	3 x	x	Labview		\$-	1	\$	-	0	\$-	
	х	Steel pipe	4464K355	\$ 7.78	1	\$ 7.78	\$-	\$ 7.7	8 x	x	Matlab		\$-	1	\$	-	0	\$-	
	х	3 port ball valv	4017T14	\$ 35.83	1	\$ 35.83	\$-	\$ 35.8	3 x	x	Solidworks		\$-	1	\$	-	0	\$-	
	x	bushing adapte	4464K399	\$ 8.56	1	\$ 8.56	\$-	\$ 8.5	6 x	x	Gantter		\$-	1	\$	-	0	\$-	
	х	Hose 1	7454T12	\$ 12.00	1	\$ 12.00	\$-	\$ 12.0	0										
	х	Hose 2	7454T15	\$ 12.33	1	\$ 12.33	\$-	\$ 12.3	3			Progress	73	% (by part #)		-	Total	\$ 7,513.74	4
	х	pipe	4813K63	\$ 79.72	1	\$ 79.72	\$-	\$ 79.7	2				34	% (by \$ Value)		I	Budget	800)0
	x	bushing adapte	4464K151	\$ 13.76	1	\$ 13.76	\$-	\$ 13.7	6							1	margin	6.078	%
	x	inline tee	44605K627	\$ 8.68	2	\$ 17.36	\$-	\$ 17.3	6								-		-
x	х	stepper motor		\$-	0	\$-	\$-	\$-	1										
		EDMing		\$ 1,100.00	1	\$ 1,100.00	\$-	\$ 1,100.0	0										
		BPRegulator		\$ 1,600.00	1	\$ 1,600.00	\$ 240.00	\$ 1,840.0	0										
		Housing block		\$ 90.00	4	\$ 360.00	\$ 54.00	\$ 414.0	0										
		gear block		\$ 90.00	2	\$ 180.00	\$ 27.00	\$ 207.0	0										
		brackets		\$ 100.00	1	\$ 100.00	\$ 15.00	\$ 115.0	0										
		coupler		\$ 25.15	2	\$ 50.30	\$ 7.55	\$ 57.8	5										
		Tooling for gea	rs	\$ 500.00	1	\$ 500.00	\$ 75.00	\$ 575.0	0										
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40













- We are slightly behind schedule (3 days 1 week)
 - Encountered problems with pump manufacturing
 - Air motor testing
 - Unable to find cheaper parts resulted in manufacturing them
- Increase in team hours to accomplish sensor and integration tasks
- Moving Forward:
 - Increase in sub-component testing from all team members
 - Timeframe: ~100 hours for Electronics
 - ~200 hours for Software
 - ~100 hours for Manufacturing/Assembly
 - ~150 hours for Testing





Questions?





Backup





- Worries about shafts wearing into the housing necessitated this solution
- Possible to post machine down the ends to press fit into bearings and key in the proper locations for the gears and coupler







• To prevent pressure pushing the driveshaft out, an extra retention method was included to prevent axial pushback since the shaft is essentially wedged between the mounted pump and mounted motor on the stand









- The geometry is being updated to optimal specs. Will be finalized/double checked by COB Tuesday
- May included a double seal to mitigate possible leaking as we enter testing

Prelim Housing Drawing with GD&T







Pump Control - Software

Pressure and temperature combined test setup

Overview

Pump



Budget







Pressure and temperature combined test main loop



SAS Phase 1 – Sensor Testing and Simulation

Electronic pressure regulator design	 Designing motor-regulator mechanical interface Designing pressure regulation through control of motor voltage
Sensor Testing	 Circuit Design: Power requirements, reading outputs Calibration: sampling rate, noise compensation, error rejection
Simulink Modeling	 Model Drive and Pump system Design control law for throttling pump flow rate Design realistic throttle profiles
LabView Design	 System circuit design Implementation of safety monitoring software

Key Results to obtained

- Completion of pressure regulating system, obtaining resulting accuracy, slew rate, power requirements
- Pressure Transducer performance
- Tachometer mounting and performance
- First iteration of electronics circuit design
- First iteration of system control law
- Estimate of system slew rate (startup and shutdown)
- Generation of various throttle profiles
- First iteration of DAQ design for both data acquisition and system control
- First iteration of system user interface
- Design of system monitoring software
- Confirming testing location

Phase 2 Functional Reqs. met – None







Drive System Testing

- Motor: RPM/Torque Test
- Motor: Helium Run Test
- Manual Regulator: Pressure Fluctuations
- Electronic Regulator: Calibration to hit maximum accuracy.
- Tachometer: Calibration to hit maximum accuracy
- Power-Off testing: Verifying that solenoid valve correctly closes in case of power loss.
- Throttling Test 1: Throttling from 10-100% using the electronic pressure regulator, confirming control with pressure transducer.
- Throttling Test 2: Fully assembling drive system components. Running through throttle profiles.

Pump System Testing

- Manual Regulator: Pressure Fluctuations
- Power-Off testing: Verifying that solenoid valve correctly closes in case of power loss.
- Electronic Back Pressure Regulator: Calibration to hit maximum accuracy

- Numerous iterations of throttling control software, monitoring software and electronic regulator software
- Assessment of unforeseen issues, correction of affected components
- Validation of component capabilities, allowing for full system assembly to occur







Timing

Water Testing – Subsystem Testing						
Component(s)	Testing to be Conducted	Man Hours	Status	required		
Solenoid Valve Shut-Off	Simulate excessive pressure and/or temperature readings to initiate safety shutdown	9 Hours : (2*4 + 2*0.5 + 2*0.5)	0%	required to prepare and		
Adapter, Quick Disconnect, Flexible Pipe, Pressure Ducer	Quantify pressure losses (water) associated with pre-piping assembly	7.5 Hours : 2*1 + 2*1.5 + 2*1	0%	assemble tes Hours: Time		
Pre-Pump pressure loss	Quantify pressure loss (water) of full assembly leading to pump inlet	10.5 Hours : 3*2 + 3*0.5 + 3*1	0%	required to conduct test		

Total Man Hours: 27

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ed to et test

Hours: Time required to disassemble test and save results

Multiplier: Repeat test for identical sensor /component







Timing

Air Testing – Subsystem Testing						
Component(s)	Testing to be Conducted	Man Hours	Status	required		
Solenoid Valve Shut-Off	Simulate excessive pressure and/or temperature readings to initiate safety shutdown	4 Hours : (2*1 + 2*0.5 + 2*0.5)	0%	Hours: Time required to prepare and		
Pressurized Gas, Manual Regulator, Solenoid Valve (Air), Relief Valve, Filter, Pressure Transducer	Quantify pressure losses (air) associated with pre-regulator assembly	10.5 Hours : 3*1.5 + 3*1+ 3*1	0%	Hours: Time required to		
Full air system excluding Air Motor	Detection of pressure regulator miss- calibration/erroneous operation	13.5 Hours : 3*2 + 3*1 + 3*1.5	0%	conduct test Hours: Time		

Total Man Hours: 28

Time required to disassemble test and save results

Multiplier: Repeat test for identical sensor /component







Timing

Personnel: Operators required

Hours: Time required to prepare and assemble test

Hours: Time required to conduct test

Hours: Time required to disassemble test and save results

Multiplier: Repeat test for identical sensor /component

Air Testing – Subsystem Testing							
Component(s)	Testing to be Conducted	Man Hours	Status				
Solenoid Valve Shut-Off	Simulate excessive pressure and/or temperature readings to initiate safety shutdown	9 Hours : (2*4 + 2*0.5 + 2*0.5)	0%				
Pressurized Gas, Manual Regulator, Solenoid Valve (Air), Relief Valve, Filter, Pressure Transducer	Quantify pressure losses (air) associated with pre-regulator assembly	7.5 Hours : 2*1 + 2*1.5 + 2*1	0%				
Full air system excluding Air Motor	Detection of pressure regulator miss- calibration/erroneous operation	10.5 Hours : 3*2 + 3*0.5 + 3*1	0%				
Full Air Assembly, air motor connected to friction assembly							

0.1

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Total Man Hours: 27

OverviewPump
ManufacturingPump
ControlTesting/
ScheduleBudget





Full scale testing to be conducted

- Drive Shaft Alignment
- Mass flow rate testing
- Throttle profile testing
- System startup and shutdown testing
- Emergency shutdown testing
- Restartability testing

- Determination of any misalignment of motor/pump driveshaft
- Pump performance: mass flow rate, back pressure, efficiency
- Observation of system under numerous throttle profiles
- Iteration on shutdown procedures to optimize restartability
- Iteration on control law to account for unaccounted system properties







Full scale testing to be conducted

- Throttle profile testing: (10 100% Throttle), slew rate testing.
- Quantifying magnitude of outlet pressure fluctuations
- Throttle profile testing (ran through various profiles)
- System startup and shutdown testing, with emergency shutdown and restart-ability

- Pump performance: mass flow rate, back pressure, efficiency
- Observation of system under numerous throttle profiles
- Iteration on shutdown procedures to optimize re-startability
- Iteration on control law to account for unaccounted system properties

SAS Standard Testing Procedure - Example

Throttle Calibration Test

- 1. Pump is started up and commanded to throttle setting
- 2. The pump is run at throttle setting
- 3. Flow enters control volume for 10 secs
- 4. Flow is diverted back to regular vent
- 5. Pump is shutdown
- 6. Control volume is measured and recorded
- 7. Test is re-iterated as needed for other throttle settings

Critical Test Elements

- Pump-Drive system
- Control Volume
- Flow Bypass system
- 3 Team membersmonitoring test

- Validation of throttle model design
- Refinement of throttle model design
- Meeting of critical project element

Safety Set-Up





Worst Case Failures

- 1. Drive system flywheel 225 J
- **2**. Drive system casing -16 J
- **3**. Pump gears -36 J

<u>Cinder Block Housing Strength</u>

- Chipping 300 J
- Cracking 600 J
- Penetration 800 J
- Failure 1000 J
- Complete destruction >1300 J



- Converting from ADC to pressure
- Find the MSMT out of the ADC to get the Voltage into the ADC
- Find the pressure out of the Pressure transducer
- Find pressure by dividing it by the voltage per pressure ratio

$$V_{ADC} = V_{out,max} \frac{MSMT}{2^{16}}$$

$$V_{Press} = \frac{V_{ADC}}{G} + V_{offset}$$

$$P = \frac{V_{Press}}{Ratio_{V2P}}$$







- Electronic pressure regulators with high volume flow are not found, but manual do exists
- Combining a manual pressure regulator with an encoder and stepper motor
- Motor shall have a minimum angular velocity of 300 RPM
- No error, and time of settle of less than 1 second



 $\theta_{M,d}$

 $\omega_{d,r}$

 $\omega_{d,r}$

 $\theta_{M,r}$

Encode

- Controlling a stepper motor based on the position
- This position will allow the output pressure





Automatic Pressure Regulator Diagram

- Control points of view for the pressure regulator
- Can be simplified by combining both plants
- Needs to be tested to find the correlation of angular position with output RPM

Pressure regulator control





Automatic Pressure Regulator





Automatic Pressure Regulator



- Simplified version of the model
- Allows to control the drive system based on the voltage input





<u>Signal Processing – Differential Amplifier</u>

- Use for creating an offset and applying a gain to the voltage out of the pressure transducer
- This will allow us to look at v₂ a range from 600 – 800 psi a with higher accuracy.







- Being able to from 28 V to .6 V
- The output will be used to provide the offset needed for V1







Low pass filter

 $\underline{\text{Vs}}$

- Easy to make
- First order filter
- Only one pole

- Harder to make
- Second order filter
- Two poles

$$T = \frac{1}{1 + sRC}$$

$$T = \frac{\omega_0^2}{s^2 + \frac{\omega_0}{Q}s + \omega_0^2}$$





- Prevents to look at high frequencies that are irrelevant
- This will have a cutoff frequency at 2 kHz










